

ME/EMA 540

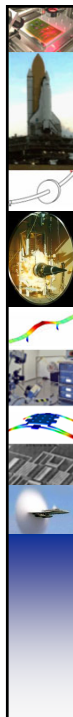
Experimental Vibrations & Dynamic System Analysis

Module #6: Operational Modal Analysis (OMA) /
Natural Excitation Technique (NExT)

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Review of EMA with Frequency Response Functions (FRFs)

- Recall that the basis of EMA is to extract modal parameters from FRFs

$$- \{X\} = [H]\{F\} \text{ or } \{X\} = \{H_p\}F_p$$

$$\{H_p(\omega)\} = \sum_{r=1}^N \frac{\{\phi_r\}\phi_{pr}}{\omega_r^2 - \omega^2 + 2i\omega\zeta_r\omega_r}$$

- And FRFs are estimated from the auto and cross spectra:

$$- H_1(\omega) = S_{XF}(\omega)S_{FF}(\omega)^{-1}$$

$$S_{XF}(\omega) = \frac{1}{N_{avg}} \sum_{j=1}^{N_{avg}} X_j(\omega)F_j(\omega)^* = E(X_jF_j^*)$$

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Operational Modal Analysis

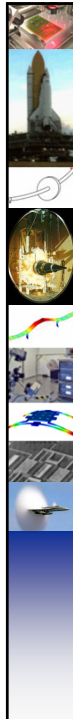
- Originally called the “Natural Excitation Technique” (NeXT)
 - by G.H. James (NASA) and T. G. Carne (Sandia)
 - G. James, T. G. Carne, and J. P. Lauffer, “The natural excitation technique (NeXT) for modal parameter extraction from operating structures,” The International Journal of Analytical and Experimental Modal Analysis, vol. 10, no. 4, p. 260, 1995.
- Annual Conference:
 - <http://iomac.eu/>

International Operational
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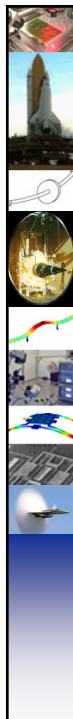


Derivation of OMA

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Operational Modal Analysis

- Summary of Derivations
 - The modes of a structure are readily apparent in the power spectrum of its response to a broadband, white, random input.
 - Peaks -> Natural frequency and damping
 - Mode shapes:

$$S_{x_j x_n}(\omega) = E(X_j X_n^*)$$

$$S_{x_j x_n}(\omega) = \sum_{r=1}^N \frac{\phi_{jr} \phi_{pr}}{\omega_r^2 - \omega^2 + i\omega 2\zeta_r \omega_r} S_{F_p F_p}(\omega) H_{np}^*$$

Near $\omega = \omega_r$,

$$S_{x_j x_n}(\omega) = \frac{\phi_{jr} S_{F_p F_p}(\omega) (\phi_{pr})^2 \phi_{nr}}{4\zeta_r^2 \omega_r^4}$$

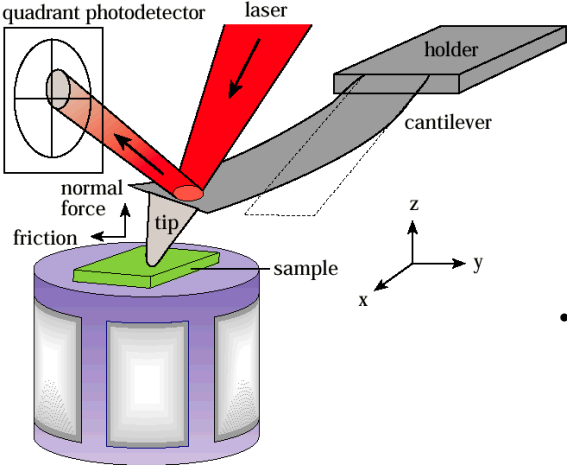
$$S_{\{x\} x_n}(\omega) = \{\phi\}_r \frac{S_{F_p F_p}(\omega) (\phi_{pr})^2 \phi_{nr}}{4\zeta_r^2 \omega_r^4}$$

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Atomic Force Microscopy

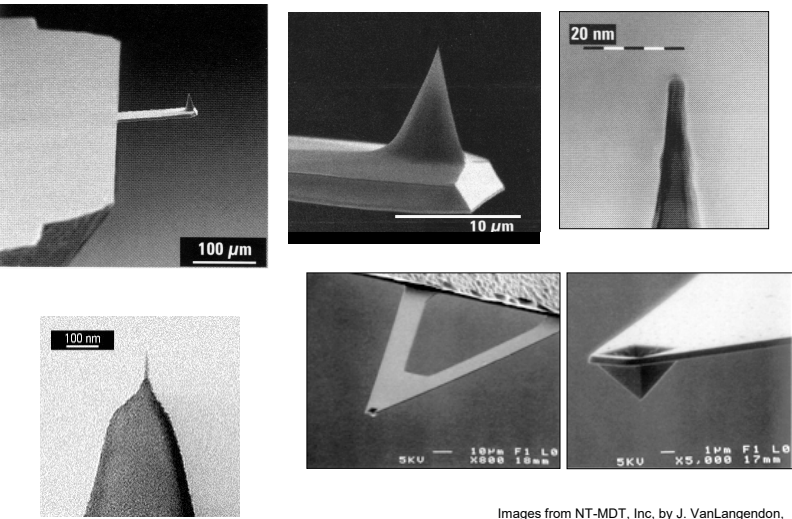


- AFM: A mechanical detection system for studying materials at the nanoscale.
 - Developed in 1986 by Binnig, Quate, and Gerber in a collaboration between IBM and Stanford University
- Laser based detection system:
 - Sub nanometer displacement resolution.
 - Sub nano-Newton force resolution.

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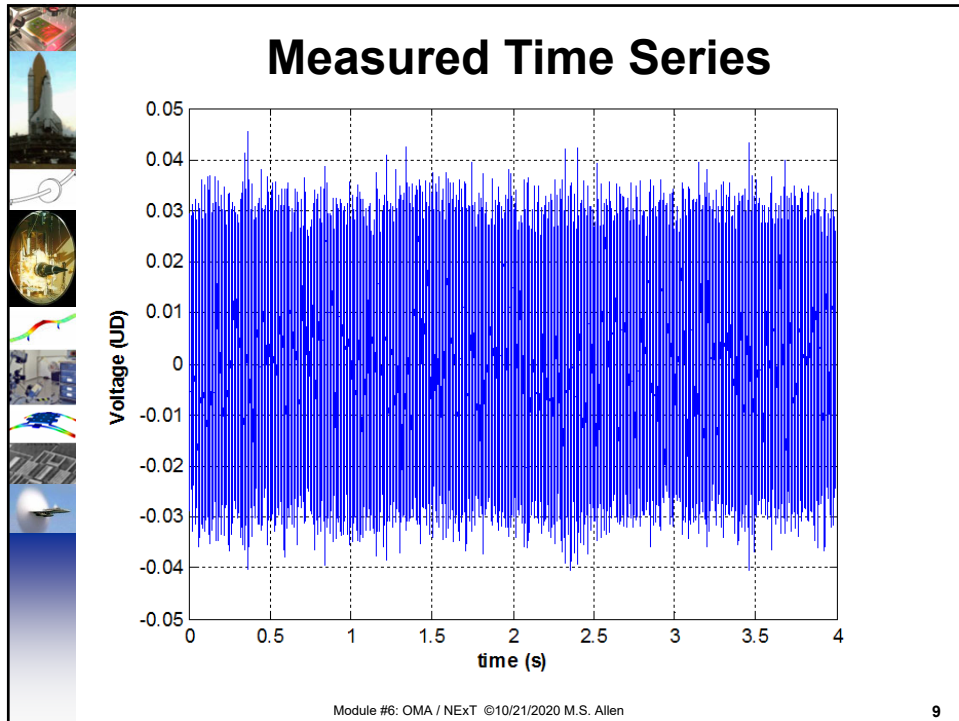
The AFM probe is a microfabricated cantilever (~100 μm) and tip (<50 nm radius)



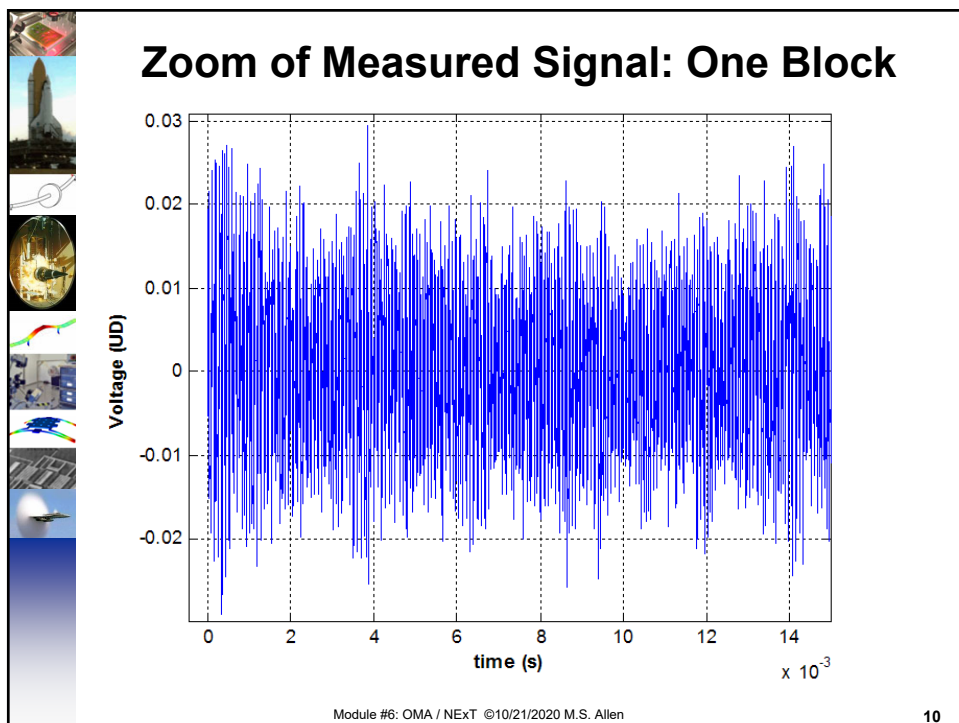
Images from NT-MDT, Inc. by J. VanLangendon,
UW-Madison and from spmtips.com.

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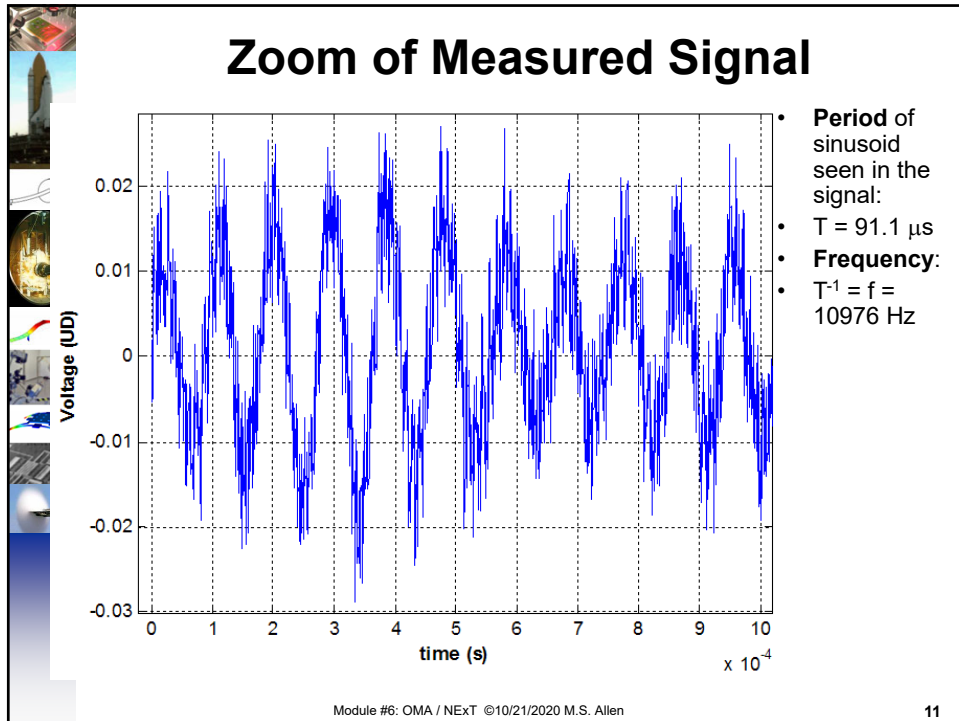
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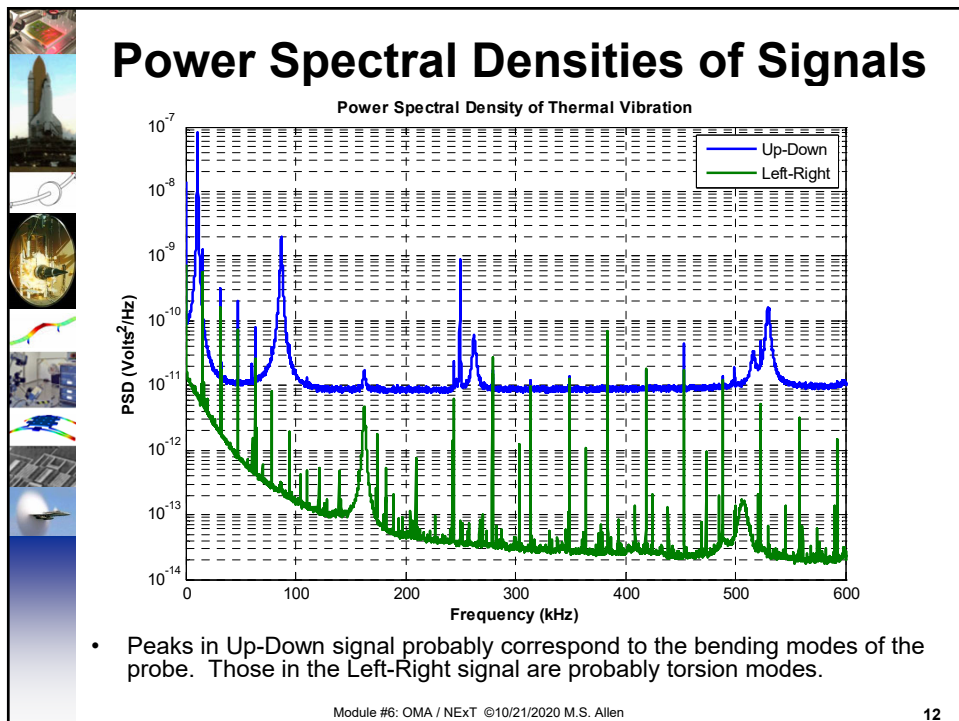
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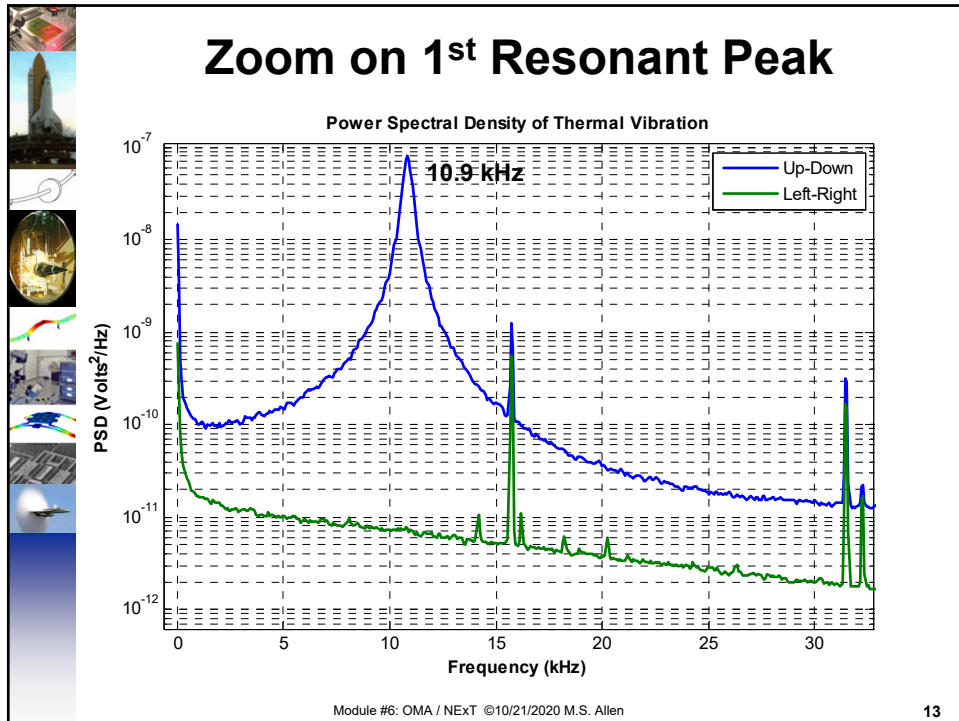
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Case Study: Wind Turbine OMA

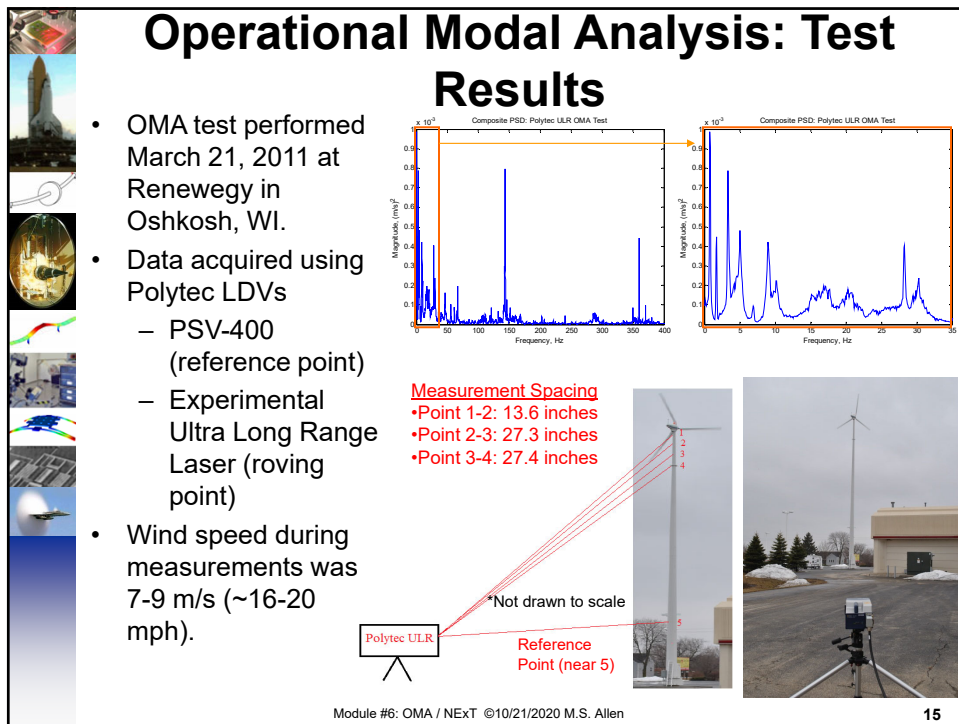
VP 20 General Layout

- 20 kW Wind Turbine
- Renewegy LLC, Oshkosh, WI
- 9.4m diameter rotor
- ~30m tower height

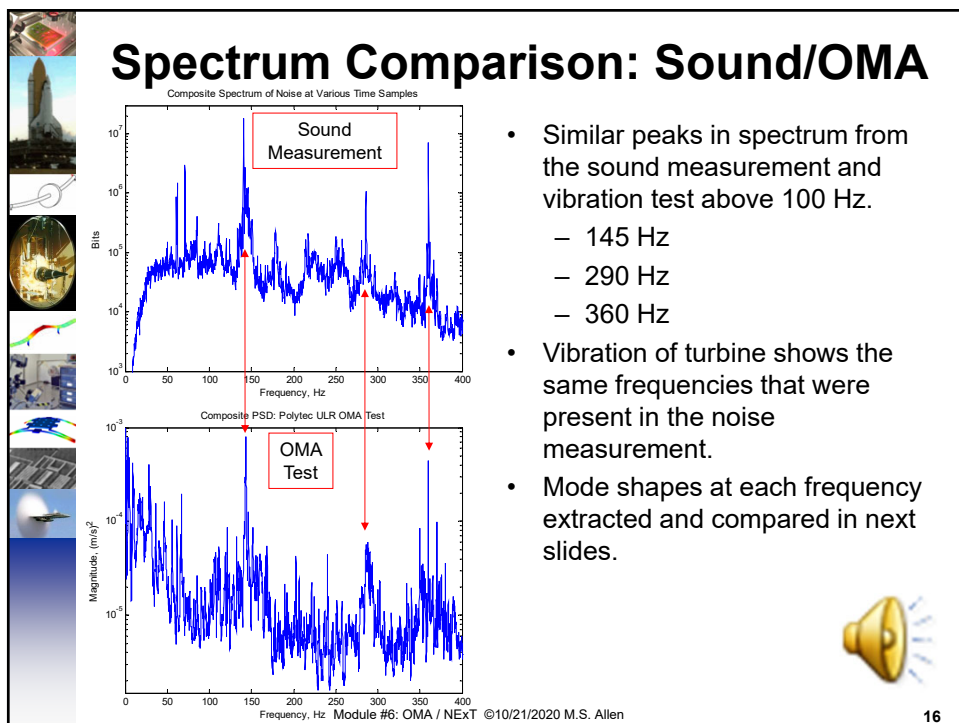
- Gearbox between Generator and Rotor so both run at different speeds.
 - Nominal Rotor Speed:
 - 100 rpm = 1.67 Hz
 - Nominal Generator Speed:
 - 600 rpm = 10 Hz

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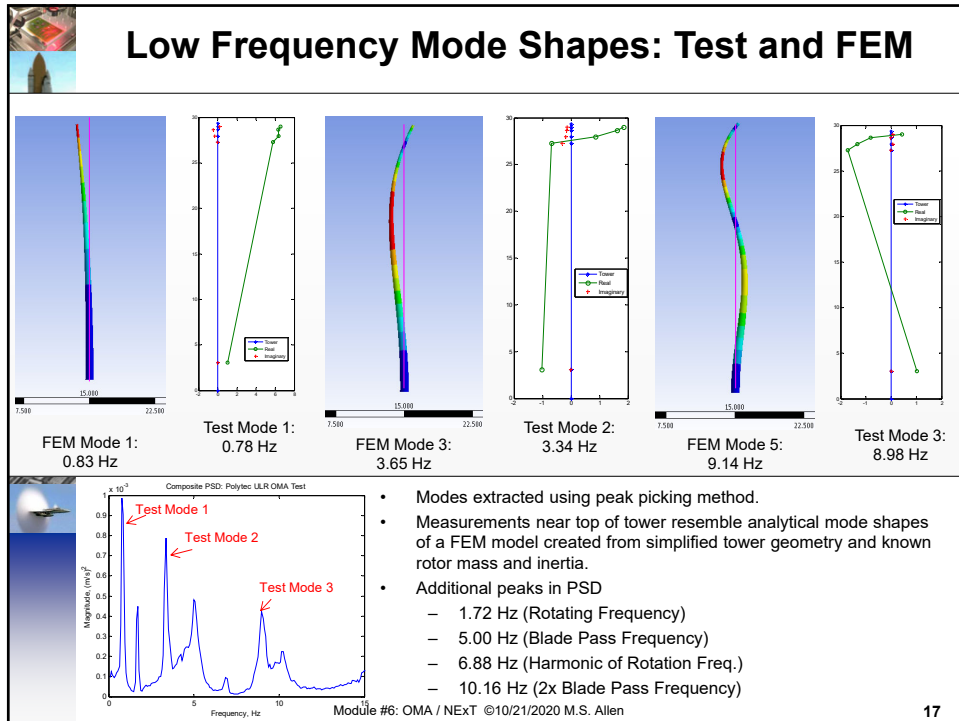
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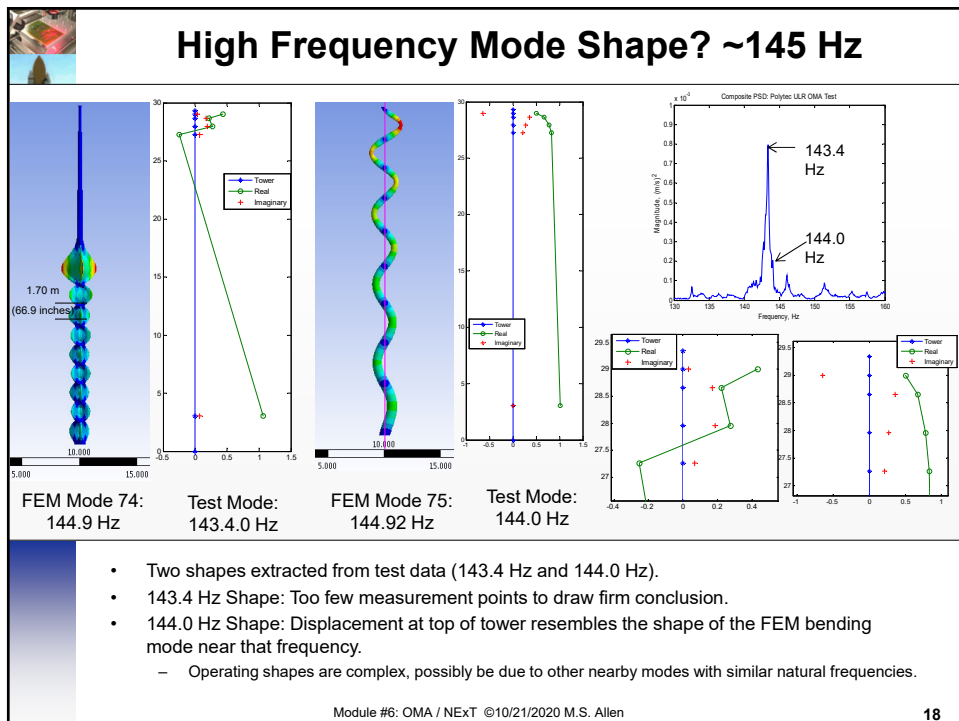
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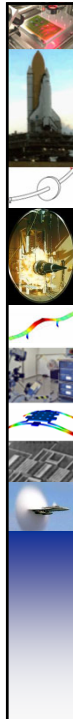
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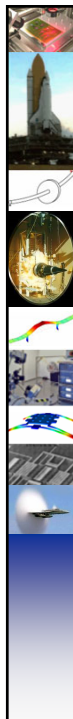


- The following slides were covered previously.

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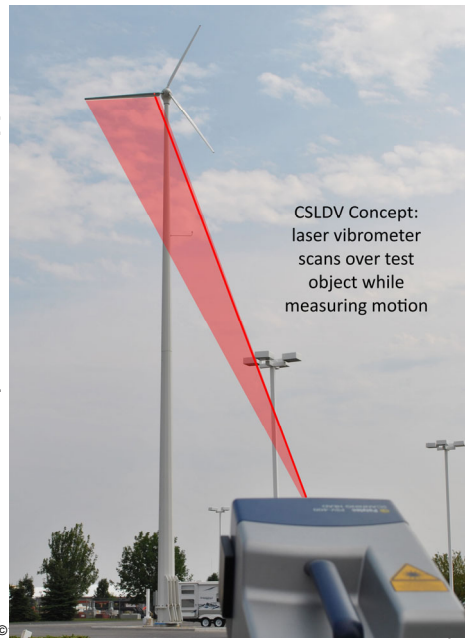
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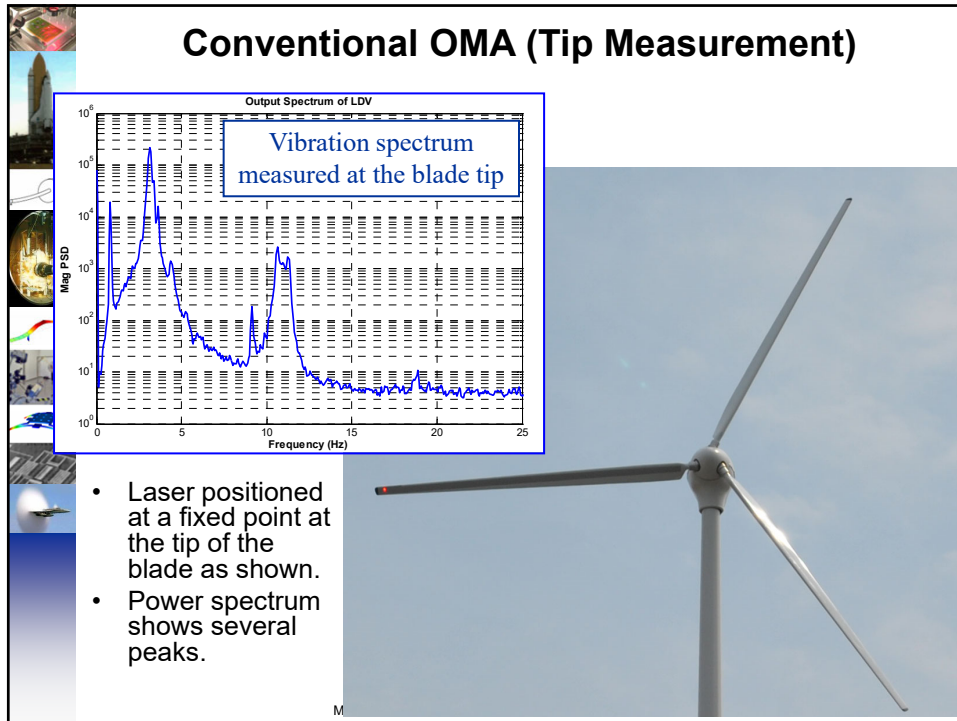
OMA-CSLDV on Wind Turbine Blade

- Alternative to obtain more detailed mode shape measurements:
 - Allen, Sracic & Yang recently extended operational modal analysis to continuous-scan measurements.
 - Laser sweeps across the surface of interest while acquiring measurements.
 - Has the potential to obtain spatial resolution that is hundreds of times better in the same amount of time.

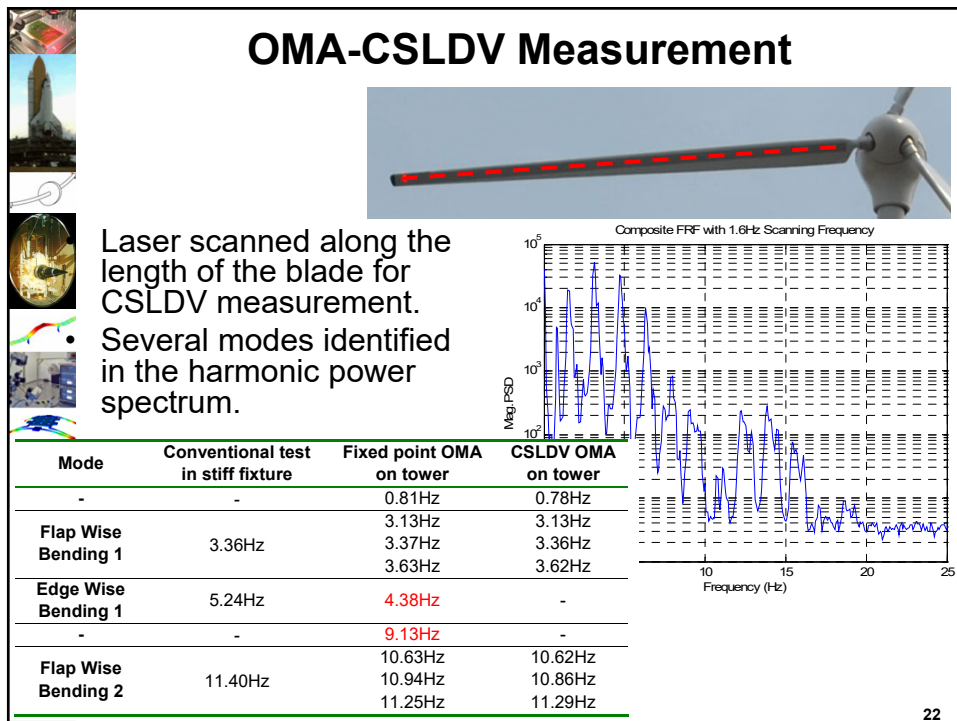


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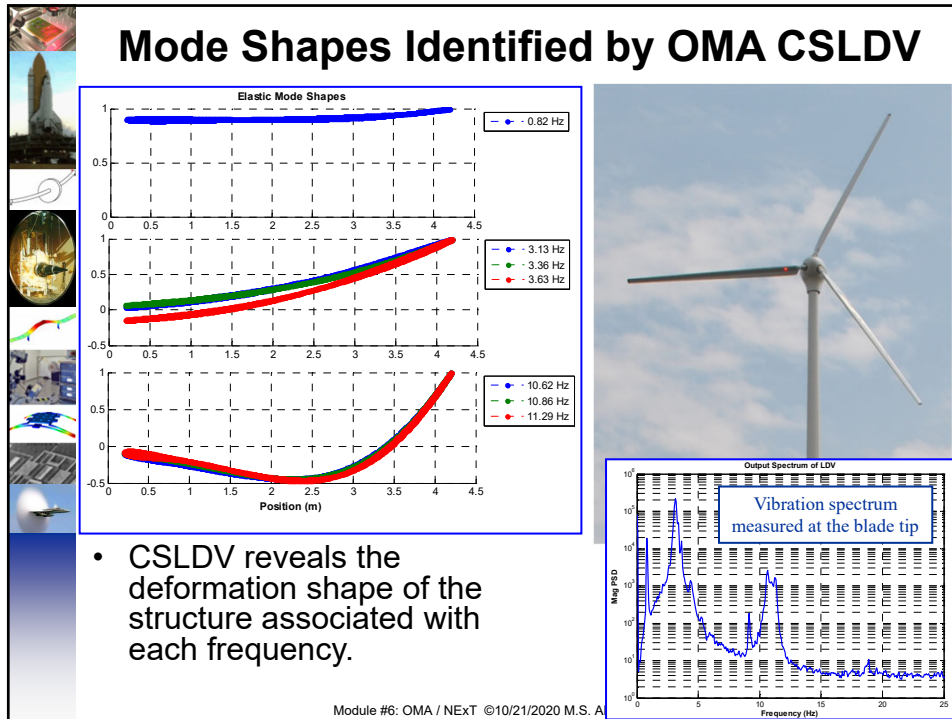
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Comparison with Conventional Methods

- Conventional Test Methods:
 - OMA with accelerometers (fixed sensors)
 - Requires attaching sensors to the points of interest and running cables to data acquisition (or wireless transmitters).
- OMA with conventional scanning LDV
 - At least two measurement points needed to obtain mode shapes.
 - Cost per LDV: \$80,000+
 - Each pair of points must be observed for at least 10 minutes.
- The results presented here were acquired with one ground based laser and two 10-min time records!

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